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(54) Venetian blinds roll-up device

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SPECIFICATION

1 1. TITLE OF THE INVENTION

2 Venetian blinds roll-up device

3 2. SCOPE OF PATENT CLAIMS

4 A venetian blinds roll-up device characterized
5 in that a constant force spring is mounted on a
6 drive shaft or operating shaft for performing the
7 rolling up of venetian blinds, and the radius of
8 curvature of the constant force spring is changed
9 in response to the gradually increasing change
10 in the load as the blinds are rolled up, so as to
11 constantly generate a spring torque corresponding
12 to the load of the blinds.

13 3. DETAILED DESCRIPTION OF THE INVENTION

14 When venetians blinds are rolled up by turning
15 the operating shaft of the blinds by means of a gear
16 mechanism or the like, the load increases gradually
17 as the roll-up progresses, and conversely when the
18 blinds are rolled down, the load becomes smaller
19 as the roll-down progresses. Therefore, the force
20 necessary to manipulate the operating shaft of the
21 blinds is not constant and changes ceaselessly.

22 The present invention relates to a venetian
23 blinds roll-up device characterized in that a
24 constant force spring is mounted on the operating
25 shaft of the venetian blinds, and the radius of
26 curvature of the constant force spring is changed in
27 response to the change in load as the blinds are
28 rolled up, so as to constantly generate a spring

2

1 torque in the opposite direction and identical to the
2 torque due to the load of the blinds acting upon the
3 operating shaft, which has the effect that the force
4 for manipulating the operating shaft in order to
5 perform roll-up and roll-down of the blinds can be
6 a: small, constant force regardless of the position
7 of the blinds, and that the blinds do not fall
8 spontaneously due to the weight of the blinds if
9 roll-up is stopped mid-way, but are rather stopped
10 at that position by the spring torque.

11 Regarding the illustrated embodiment
12 example, an operating shaft 2 is mounted
13 horizontally on the upper case 1 of the venetian
14 blinds, tapes 4, 4 are wound onto drums 3, 3
15 attached to multiple locations on the operating
16 shaft 2, and each tape 4 is passed through through-
17 holes in the slats 5, 5 of the blinds and is coupled
18 to lower case 6. A bevel gear 7 is attached to one
19 end of operating shaft 2, a chain pulley 10 is fixed
20 to drive shaft 9 of bevel gear 8 which engages with
21 bevel gear 7, and an endless chain 11 is hung on
22 chain pulley 10 and suspended downward.

1 Conventional venetian blinds are configured as
2 described above, whereby one side of chain 11 is
3 pulled to tum the chain pulley 10, the operating
4 shaft 2 is turned through engagement and
5 interlocking of the bevel gears 8 and 7, drums 3, 3
6 tum in one direction to pull up the lower case 6 for
7 example by winding tapes 4 around the drums 3,
8 and the slats 5, 5 are sequentially stacked on the
9 lower case 6 and rolled up. Therefore, at the start
10 of the roll-up, there is only the load of the lower
11 case 6, but as the roll-up progresses, the load of the
12 slats 5 is applied, so the torque to be applied in
13 order to tum the operating shaft 2 is not constant
14 and changes ceaselessly.

15 In the present invention, regardless of loads,
16 for the purpose of performing the turning of the
17 operating shaft 2 with a small constant force, as
18 shown in FIG. 1 and FIG. 2, a constant force
19 spring is mounted on the operating shaft 2, or as
20 shown in FIG. 3 and FIG. 4, a constant force
21 spring is mounted on drive shaft 9 which turns the
22 operating shaft 2. In FIG. 2, support plate 12 is
23 mounted inside the upper case 1, a shaft tube 13,
24 which allows the operating shaft 2 to pass through
25 and wedges it in place is secured to the support
26 plate 12 (FIG. 6), and a constant force spring 17 is
27 wound diagonally between drum 14 fixed to shaft
28 tube 13 and drum 16 which is rotatably axially
29 fitted onto center shaft 15 provided on support
30 plate 12. Thus, in response to the torque of turning
31 the operating shaft 2 based on the load P as the
32 blinds are rolled up to the uppermost end, as
33 shown in FIG. 9, the constant force spring 17 is
34 made to produce a torque T in the opposite
35 direction and identical to the torque due to the load
36 P by making curvature radius R small at the
37 starting end wound onto drum 14, as shown in
38 FIG. 8, and as the blinds are rolled down, the load
39 P decreases gradually, changing gradually to load
40 p when the blinds are rolled down to the
41 lowermost end, while the curvature radius of the
42 constant force spring 17 is gradually increased
43 from the curvature radius R of the starting end to
44 the curvature radius r of the terminal end shown in
45 FIG. 7, so that a spring torque t in the opposite
46 direction and identical to the torque on the
47 operating shaft 2 due to load p is generated at the
48 terminal end.

49 When a constant force spring 17 of this sort is
50 installed, it suffices to apply a small constant force
51 in order to manipulate the endless chain 11 via
52 bevel gears 7, 8 in order to tum the operating shaft
53 2, and if the manipulation of the endless chain 11

1 is stopped mid-way during roll-up or roll-down of
2 the blinds, the torque of the blinds descending
3 under their own weight will be balanced out by an
4 identical spring torque in the opposite direction
5 due to the constant force spring, so the blinds will
6 stop at the mid-way position.

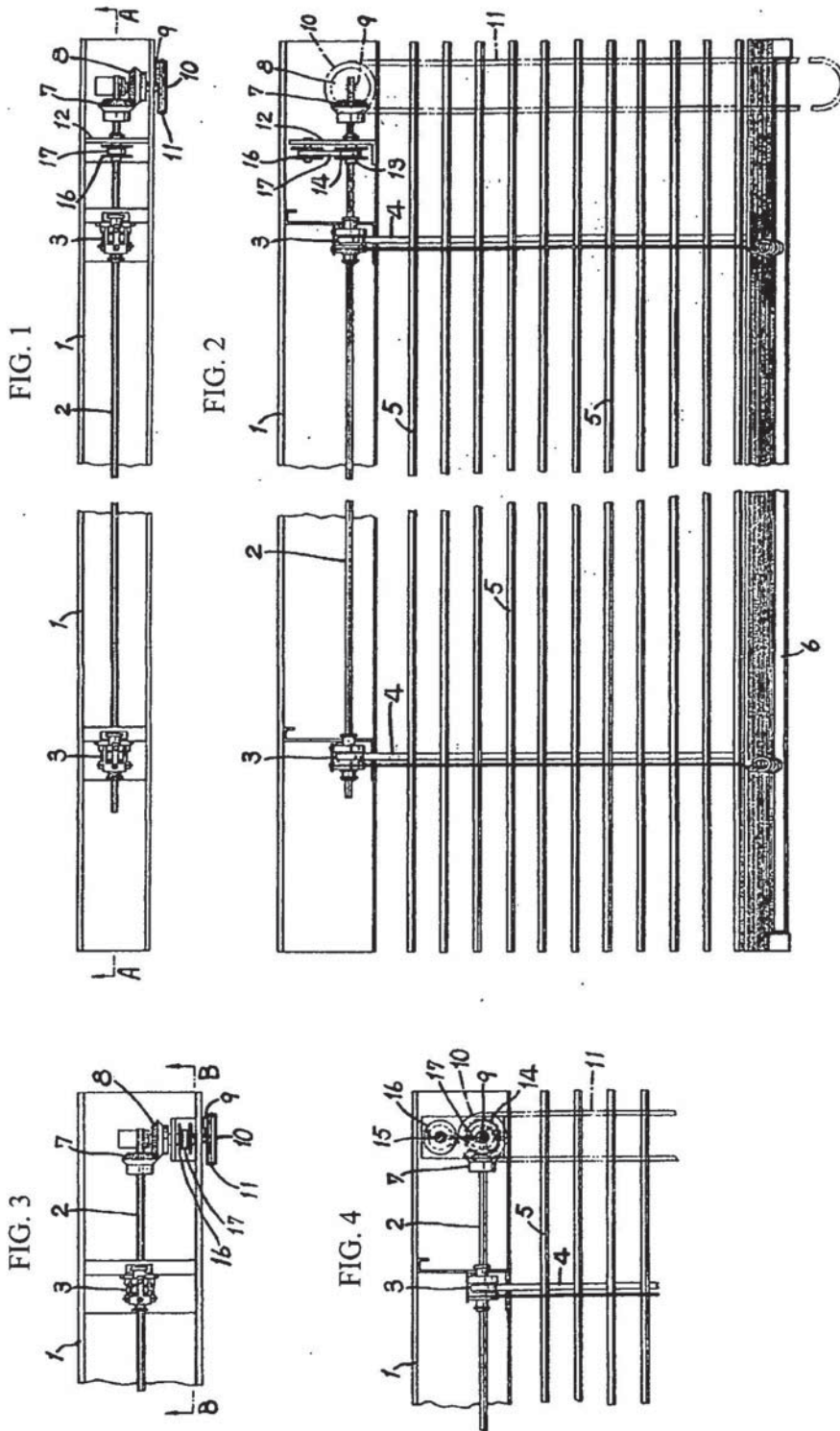
7 It will be noted that while it is ideal to change
8 the radius of curvature of the constant force spring
9 17 in response to change in load of the blinds such
10 that its torque changes gradually from torque T to
11 torque t , as shown in FIG. 9, there is no practical
12 impediment to employing a constant force spring
13 machined such that the radius of curvature
14 becomes a fixed (value) R so as to provide a spring
15 torque P in a suitable range from the starting end,
16 and whereof the radius of curvature becomes a
17 fixed (value) r so as to provide a spring torque p at
18 the terminal end, as shown by the dashed line in
19 FIG. 9.

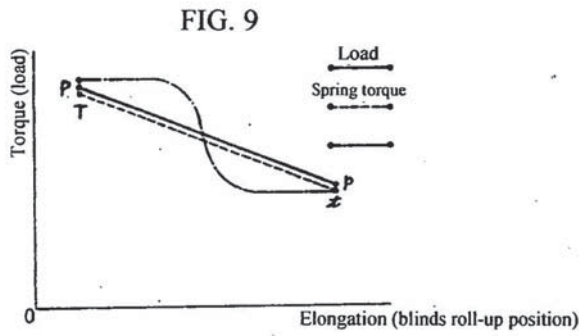
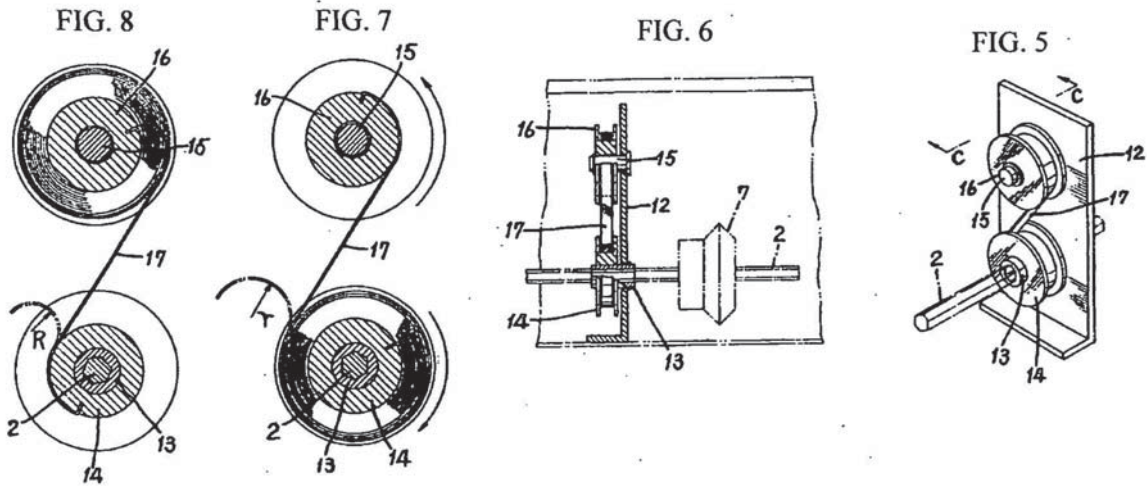
20 The effect of the constant force spring 17 in
21 FIG. 3 and FIG. 4, where it is mounted on drive
22 shaft 9, is the same as above, and the constant
23 force spring 17 can simply be wound diagonally
24 between drum 14 fixed to drive shaft 9 and drum
25 16, which is rotatably axially fitted onto center
26 shaft 15.

27 4. BRIEF DESCRIPTION OF THE DRAWINGS

28 FIG. 1 is a plan view of venetian blinds
29 embodying the present invention; FIG. 2 is a
30 vertical front cross-sectional view sectioned along
31 line A-A of FIG. 1; FIG. 3 is a partial plan view of
32 an embodiment example in which a constant force
33 spring is mounted on the drive shaft of the
34 venetian blinds; FIG. 4 is a partial front view
35 vertically sectioned along line B-B of FIG. 3; FIG.
36 5 is a perspective view of a constant force spring;
37 FIG. 6 is a vertical cross-sectional view along C-C
38 of FIG. 5; FIG. 7 is a front view showing the state
39 of the constant force spring at the position where
40 the blinds have been fully lowered; FIG. 8 is a
41 front view showing the state of the constant force
42 spring at the position where the blinds have been
43 fully raised and their load is the greatest; FIG. 9 is
44 a graph showing the change in load of the blinds
45 and the change in spring torque of the constant
46 force spring.

47 1 ... upper case of blinds; 2 ... operating shaft;
48 3 ... drum; 4 ... tape; 5 ... slat; 6 ... lower case; 7,8
49 ... bevel gear; 9 ... drive shaft; 10 ... chain pulley;
50 11 ... endless chain; 12 ... support plate; 13 ...
51 shaft tube; 14 ... drum; 15 ... center shaft; 16 ...
52 drum; 17 ... constant force spring.





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⑭ベネシャンブラインドの巻上装置

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⑰特 願 昭52—105456

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⑱出 願 昭52(1977)9月2日

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明 細 書

1. 発明の名称 ベネシャンブラインドの巻上装置

2. 特許請求の範囲

ベネシャンブラインドの巻上げを行う作動軸又は駆動軸に定荷重ばねを装着し、ブラインドを巻上げるに従い、次第に大きくなる荷重の変化に対応して定荷重ばねの曲率半径を変化して常にブラインドの荷重に相当するばねトルクを生ずるようにしてなることを特徴とするベネシャンブラインドの巻上装置。

3. 発明の詳細な説明

ベネシャンブラインドの作動軸を歯車機構等をもつて回転してブラインドを巻上げるとき、巻上げの進行と共に次第に荷重が大きくなり、逆に巻下げるとき巻下げの進行と共に荷重が小さくなる。従つてブラインドの作動軸を操作するのに要する力は一定でなく絶えず変化している。

本発明はベネシャンブラインドの作動軸に定荷重ばねを装着し、ブラインドを巻上げるとき

の荷重の変化に対応して定荷重ばねの曲率半径を変化して作動軸に作用するブラインドの荷重によるトルクと同一にして逆方向のばねトルクを生ずるようにしてなることを特徴とするベネシャンブラインドの巻上装置に関するものであつて、ブラインドの巻上げ巻下しを行うために作動軸を操作する力はブラインドの位置に関係なく一定の軽少の力でよく、さらにブラインドの巻上げを途中で中止してもブラインドの重量で自然落下せず、ばねトルクをもつてその位置に停止させることができる効果がある。

図面の実施例について、ベネシャンブラインドの上函1に作動軸2を横架し、作動軸2の数个所に取着けたドラム3、3にテープ4、4を巻回し、各テープ4はブラインドのスラット5、5…の透孔を貫通して下函6に連結する。作動軸2は一端に傘歯車7を取着け、傘歯車7に噛合する傘歯車8の駆動軸9にチェーン車10を固着し、チェーン車10に無端チェーン11を懸架して下方に垂下する。

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